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Automatic Control 2: Solutions

Exercise 1 (10 points)

The Bode form of P(s) is the following

$$P(s) = \frac{5}{48000} \frac{(1 + \frac{1}{5}s)(1 + s)}{(1 + \frac{1}{4}s)^2(1 + \frac{1}{30}s)(1 + \frac{1}{100}s)}$$

A table of the magnitude and phase contribution of the basic components of P(s) is the following

	Magnitude dB	Phase deg
K_B	$20 \log_{10}(5/48000) = -79.64$	0
(s+5)	$+20 \text{ dB/dec}, \omega_{z_1} = 5$	$+\frac{\pi}{2}$
(s+1)	$+20 \text{ dB/dec}, \omega_{z_2} = 1$	$+\frac{\pi}{2}$
$(s+4)^2$	$-40 \text{ dB/dec}, \omega_{p_1} = 4$	$-\pi$
(s + 30)	$-20 \text{ dB/dec}, \omega_{p_2} = 30$	$-\frac{\pi}{2}$
(s + 100)	$-20 \text{ dB/dec}, \omega_{p_3} = 100$	$-\frac{\pi}{2}$

The real Bode diagram is depicted in the following Figure



The Nyquist diagram is shown in the following figure

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As the nominal plant is asymptotically stable, exploiting the Nyquist criterion, the stability of the closed loop is ensured iff there are no (clockwise) encirclements of the point -1, while moving the point -1/K. The closed loop is stable iff K > 0.

Exercise 2 (10 points)

The design specifications can be translated as follows:

- $M_r \simeq 1.5$ dB By recalling the approximate formulae $M_p = \frac{2.3 - M_r}{1.25}$, where M_r is not in dB, the desired phase margin is $M_p \simeq 51$ deg
- $t_r \leq 0.01$ s Since $t_r B_3 \simeq 3$ and $\omega_c = [0.5...0.8] B_3$, assuming $t_r = 0.01$ s, the desired bandwidth is $B_3 = 300$ dB and $\omega_c = [150...240]$ rad/s. A good choice is $\omega_c = 180$ rad/s

By looking at the phase and the magnitude of $P(j\omega_c)$, the gap to compensate can be calculated as follows

 $\angle (P(j180)) = -176.81 \text{ deg}, |P(j180)|_{dB} = -88.63 \text{ dB}$

$$\Delta M = 0 \text{ dB} - (-88.63) \text{ dB}, \ \Delta \phi = M_p - (180 - 176.81) \simeq 48 \text{ deg}$$

In order to satisfy the phase specification, we choose a *lead* network with $\alpha_1 = 0.15$, centered in w_c for which $\tau_1 = \frac{1}{w_c \sqrt{\alpha_1}} = 0.0143$. This network increases the phase for about 48 deg and the magnitude for about 1.35 dB.

The phase specification is satisfied, while the new magnitude specification is $\Delta M' = \Delta M - 1.35 \simeq 80$ dB. In order to shift the magnitude diagram while preserving the phase diagram at $\omega_c = 180$ rad/s, four *lead* network are chosen at $\omega_{low} = \omega_c/1000$ rad/s, with $\alpha_2 = 0.1$ and $\tau_2 = \frac{1}{w_{low}\sqrt{\alpha_2}} = 17.5682$. The corresponding network functions have the form

$$C_{lead}(s) = \frac{1 + \tau_i s}{1 + \alpha_i \tau_i s}, \quad i = 1, 2$$

At this point, the design specifications are verified for the closed loop transfer function

$$F(s) = \frac{L(s)}{1 + L(s)}$$
, where $L(s) = C_{lead_1}(s)C_{lead_2}(s)^4G(s)$

In the following Figure is shown the Bode diagram of L(s)



Exercise 3 (8 points)

The resulting autonomous system for u(t) = 0 is

$$\dot{x}(t) = -x(t) - x^3(t)$$

for which the origin is an equilibrium point. Taking for instance $V(x) = x^2/2$, which is positive definite for all $x \in \mathbb{R}$ (i.e. $V(x) > 0 \ \forall x \neq 0$, V(0) = 0), one obtains $\dot{V}(x) = \frac{\partial V(x)}{\partial x}\dot{x} = x(-x-x^3) = -x^2 - x^4$, which is negative definite for all $x \in \mathbb{R}$ (i.e. $\dot{V}(x) < 0 \ \forall x \neq 0$, $\dot{V}(0) = 0$). Therefore, V(x) is a Lyapunov function for the given system, which proves that the origin is an asymptotically stable equilibrium point with domain of attraction equal to \mathbb{R} .

As for the feedback linearization, the system can be represented as $\dot{x}(t) = f(x(t)) + g(x(t))u(t)$, with $f(x(t)) = -x(t) - x^3(t)$, and g(x(t)) = sin(x(t)) + 2. Defining v(t) = f(x(t)) + g(x(t))u(t) one obtains

$$\dot{x}(t) = v(t)$$

which is a linear system (more precisely, an integrator). A scalar system with the only pole in -1 is $\dot{x}(t) = -x(t)$, and then it is sufficient to define v(t) = -x(t) as the state-feedback control law (i.e. v(t) = Kx(t),

with K = -1). The resulting control law $u(t) = \frac{v(t) - f(x(t))}{g(x(t))}$ is

$$u(t) = \frac{x^3(t)}{2 + \sin(x(t))}$$

Another choice leading to an easy solution is to take $f(x(t)) = -x^3(t)$, and g(x(t)) = sin(x(t)) + 2. Defining v(t) = f(x(t)) + g(x(t))u(t) one obtains

$$\dot{x}(t) = -x(t) + v(t)$$

which is again a linear system. To obtain $\dot{x}(t) = -x(t)$ one can define v(t) = 0 as the state-feedback control law (i.e. K = 0). The resulting control law $u(t) = \frac{v(t) - f(x(t))}{g(x(t))}$ is clearly the same as with the previous choice:

$$u(t) = \frac{x^3(t)}{2 + \sin(x(t))}$$

Exercise 4 (3 points)

For the given nonlinear system, let $V : \mathbb{R}^n \to \mathbb{R}$ be positive definite in a ball B_{ϵ} around the origin, $\epsilon > 0$, $V \in C^1(\mathbb{R})$. If the function

$$\dot{V}(x) = \nabla V(x)'\dot{x} = \nabla V(x)'f(x)$$

is negative definite on B_{ϵ} , then the origin is an asymptotically stable equilibrium point with domain of attraction B_{ϵ} ($\lim_{t\to+\infty} x(t) = 0$ for all $x(0) \in B_{\epsilon}$). Such a function $V : \mathbb{R}^n \to \mathbb{R}$ is called a Lyapunov function for the system $\dot{x} = f(x)$.

Therefore, the existence of a Lyapunov function is a sufficient condition for the asymptotic stability of an equilibrium point.