

UNIVERSITY OF TRENTO

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Automatic Control 2: Solutions

Exercise 1 (10 points)

The Bode form of P(s) is the following

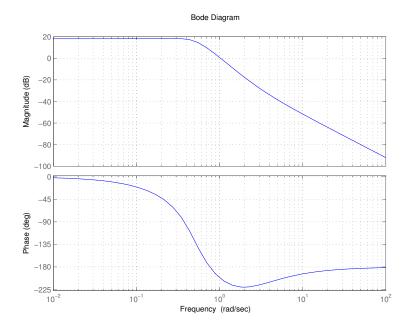
$$P(s) = \frac{8(s/2+1)^3}{(s+1)^3(4s^2+2s+1)}$$

The second order polynomial is a complex and conjugate pair of poles, then, for the polynomial identity principle, $\omega_n = 0.5$ and $\zeta = 0.5$.

For the sake of simplicity, a table of the magnitude and phase contribution of the basic components is depicted in the following

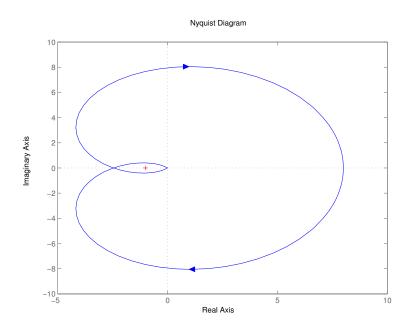
	Magnitude dB	Phase deg
K_B	$20 \log_{10}(8) = 18.1$	0
$(s+2)^3$	$+60 \text{ dB/dec}, \omega_z = 2$	$+\frac{3}{2}\pi$
$(s+1)^3$	$-60 \text{ dB/dec}, \omega_p = 1$	$-\frac{3}{2}\pi$
$(4s^2 + 2s + 1)$	$-40 \text{ dB/dec}, \omega_n = 0.5$	$-\pi$

The real Bode diagram is depicted in the following Figure



The Nyquist diagram is shown in the following Figure

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As the nominal plant is asymptotically stable, exploiting the Nyquist criterion, the stability of the closed loop is ensured iff there are no (clockwise) encirclements of the point -1, while moving the point -1/K. The closed loop is stable iff $-\frac{1}{K} < -2.46$, or, conveniently, $K < \frac{1}{2.46}$.

Exercise 2 (10 points)

The design specifications must be translated as follows.

• $M_r \simeq 1.65 \text{ dB}$

By recovering the approximate formulae $M_p = \frac{2.3 - M_r}{1.25}$, where M_r is not in dB, the desired phase margin is $M_p \simeq 50 \deg$

• $t_r \le 2.5 \text{ s}$ Since $t_r B_3 \simeq 3$ and $\omega_c = [0.5...0.8] B_3$, assuming $t_r = 2.5$ s, the desired bandwidth is $B_3 = 1.2$ dB and $\omega_c = [0.6 \dots 0.96]$. A good choice is $\omega_c = 0.6$ rad/s

In order to satisfy the steady state specification, since the P(s) is of type 0, there is no need to add integrator. Hence K_c can be chosen as $K_c = 1/(e_d K_B)$, where K_B is the dc-gain of P(s). In this case $K_c = 0.3846$. By looking at the phase and the magnitude of $P(j\omega_c)$, the gap to compensate can be calculated as follows

$$\angle (K_c P(j6)) = -152 \text{ deg}, |K_c P(j6)|_{dB} = 4.7 \text{ dB}$$

$$\Delta M = 0 \text{ dB} - 4.7 \text{ dB}, \ \Delta \phi = M_p - (180 - 152) \simeq 22 \text{ deg}$$

In order to satisfy the phase specification, a lead network with $\alpha_1 = 0.45$, centered in w_c for which $\tau_1 =$ $\frac{1}{w_c\sqrt{\alpha_1}} = 2.4845$. This network will increase the phase for about 23 deg and the magnitude for about 3.5 dB. The phase specification is satisfied, while the new magnitude specification is $\Delta M' = \Delta M - 3.5 = -8.2$ dB. In order to shift the magnitude diagram while preserving the phase diagram at $\omega_c = 0.6$ rad/s, a *lag* network is posed at $\omega_{low} = \omega_c/1000$ rad/s, with $\alpha_2 = 0.4$ and $\tau_2 = \frac{1}{w_{low}\sqrt{\alpha_2}} = 2635$.

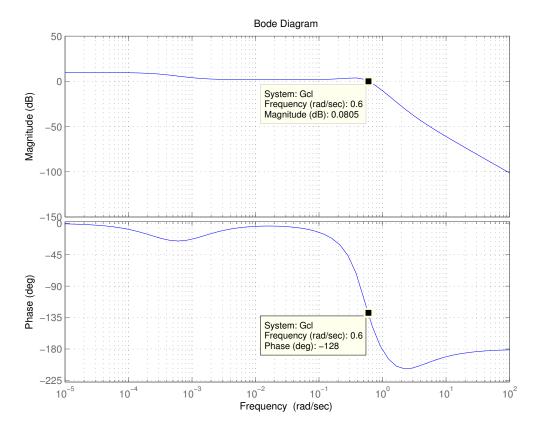
The corresponding network functions have the form

$$C_{lead}(s) = \frac{1 + \tau_1 s}{1 + \alpha_1 \tau_1 s} \quad , \quad C_{lag}(s) = \frac{1 + \alpha_2 \tau_2 s}{1 + \tau_2 s}$$

At this point, the design specifications are verified for the closed loop transfer function

$$F(s) = \frac{L(s)}{1 + L(s)}, \text{ where } L(s) = K_c C_{lead}(s) C_{lag}(s) G(s)$$

In the following Figure is shown the step response of the closed loop F(s)



Exercise 3 (6 points)

Setting V(x) = x'Px, with P = P' > 0, and imposing the decreasing of V(x), one obtains that $\Delta V(x) =$ V(x(k+1)) - V(x(k)) is negative definite if and only if for some Q = Q' > 0

$$A'PA - P = -Q$$

Now, we take Q = I, and defining $P = \begin{bmatrix} p_{11} & p_{12} \\ p_{12} & p_{22} \end{bmatrix}$ we obtain $\begin{bmatrix} p_{11}/4 + p_{12} + p_{22} & p_{12}/4 + p_{22}/2 \\ p_{12}/4 + p_{22}/2 & p_{22}/4 \end{bmatrix} = \begin{bmatrix} -1 & 0 \\ 0 & -1 \end{bmatrix}$

leading to

$$P = \left[\begin{array}{cc} -20 & 8\\ 8 & -4 \end{array} \right]$$

which is positive definite. Therefore, the origin is a globally asymptotically stable equilibrium point for the system.

Exercise 4 (7 points)

• Given the discrete-time system of order n

$$x(k+1) = Ax(k) + Bu(k)$$
$$y(k) = Cx(k) + Du(k)$$

a deadbeat controller is a state feedback controller u(k) = Kx(k) which places all the closed-loop poles at the origin, i.e. $det(zI - A - BK) = z^n$. Since by Cayley-Hamilton theorem $(A + BK)^n$, the state vanishes after n steps: $x(n) = (A + BK)^n x(0) = 0$, and then remains at the origin. The procedure to obtain the vector K is exactly the same as in an eigenvalue assignment problem.

• System identification is a procedure to build a mathematical model of the dynamics of a system from measured data. Such procedure is often used in practice because the dynamical model is difficult to obtain due to the complexity of the system and/or to lack of knowledge on it. Also, a first principle model is sometime too complex for the design of a controller.

Three kinds of models: white box (model structure based on first principles, model parameters estimated from measured data); grey box (model structure partially known from first principles, the rest reconstructed from data), black box (model structure and parameters unknown, all reconstructed from measured I/O data).